

# Numerical Study of a Planar Micromixer with Circular and Fin Obstacles

Md Readul Mahmud

Independent University, Bangladesh, Department of Physical Sciences  
Plot 16, Block B, Aftabuddin Ahmed Road  
Bashundhara Residential Area, Dhaka 1229  
mahmud.readul@iub.edu.bd

**Abstract** - The design and characterization of a passive planar O mixer with two different kinds of barriers to improve mixing performance are reported in this study. The computational fluid dynamics (CFD) program ANSYS 15 is used to perform computational studies on mixing and fluid flow in microchannels over a broad range of Reynolds numbers, from 1 to 100. The outcomes demonstrate that the O mixer with obstacles performs significantly better at mixing than the O mixer without obstacles. The explanation is that obstacles cause the fluid path length to increase, giving the fluids more time to diffuse. The O mixer with circular and fin obstacles is three times more efficient than the O mixer without obstacles in all scenarios. Due to the absence of any obstructions within the channel, the O mixer has the lowest pressure drop. The O mixer with circular & fin obstacles is the most economical one since it has the lowest mixing cost, which is a crucial feature for incorporation into intricate, cascading microfluidic systems. The proposed O mixer with obstacles can be easily manufactured and integrated into devices for a variety of macromixing applications due to its low mixed cost and simple planar construction.

**Keywords:** CFD, Efficiency, Micromixer, Mixing cost, Planar mixer.

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## 1. Introduction

Mixing various substances is a typical act of regular day-to-day activity yet it is generally difficult to accomplish good or homogeneous blending.

Microdevices and micromixers serve the purpose of obtaining excellent mixing on a micro-scale [1]. Micromixers have a high surface-to-volume proportion because of their small dimension, which is a defining characteristic compared to conventional-size chemical process equipment [2]. The flow inside the micromixer is usually laminar due to its small size and mixing usually depends on molecular diffusion at a low Reynolds number [3]. Therefore, good mixing takes a long time and a long channel length. The application of microdevices and micromixers is increasing daily in various applications such as the chemical industry, biomedical industry, and biochemical fields [4-6]. Process safety, inexpensive production expenses, less chemical and reagent consumption, enhanced process control, faster process optimization, simple implementation, and easy scale-up by "numbering up" are just a few of the many benefits of micromixers [7-11].

Mixers are broadly divided into two types, active and passive [12]. There are always active parts in active mixers to achieve excellent mixing [13]. On the other hand, passive mixers utilize various channel sizes and lengths, and unique geometric configurations to compensate for the absence of active elements [14]. The passive mixer increases the contact area between fluids and promotes molecular diffusion. The primary categories of active mixers include thermal, time-pulse, acoustic, magnetic, electrodynamic, dielectrophoretic, pressure perturbation, and other varieties [15-16].

T shape and Y shape mixers are the oldest mixers designed and analyzed by many researchers in recent years [17-21]. Numerical and/or experimental flow regimes, the influence of secondary flow, vortex flow,

and mixing performance have been computed extensively in recent years [22-24]. Generally, the T mixer provides poor efficacy at low Reynolds numbers due to the laminar nature of flow (also the absence of advection). Hence different kinds of obstacles and barriers are placed inside the channel to create chaotic advection and as a result, increase the efficiency. Many authors introduce various sizes and shapes of obstacles in T mixers which increase efficacy but result in high-pressure drop [25-28]. Four passive micromixers (G1, G2, G3, and G4) were studied but G1 and G4 designs provided a high mixing due to the presence of chaotic advection [29]. A T mixer having staggered fins has been numerically studied for a set of parametric (spacing of fins, angle of inclination, Reynolds number, and width of fins) [30]. M. Nimafar et. al. [31] reports a basic O-type mixer for low Reynolds numbers ( $0.08 < Re < 4.16$ ) and after 15 mm along the channel length, the experimental mixing efficiency is about 81% and 17.6% at  $Re = 0.803$  and  $Re = 4.166$ , respectively.

Several groups have studied mixers introducing barriers and baffles that modify the overall channel geometry [32-34]. A. A. S. Bhagat et al. [35] used circular, triangular, and diamond impediments to boost efficiency at  $Re = 1$ . A range of diamond obstacles was added to the designs by Both Shim et. al. [36] and Chung et. al. [37], resulting in good efficiency at  $Re = 200$ .

In recent years, ridges and grooves have been introduced to achieve chaotic mixing. In chaotic-advection micromixers, 3D channel structure [38-39], and planar design [40] were used to enhance fluid mixing. A. D. Stroock et. al. [41] proposed herringbone-shaped grooves which yield high efficiency (90%) at a low Reynolds number.

In this present work, a simple O-shaped mixer is studied numerically. To improve the performance of the O mixer circular obstacles and a combination of circular & fin obstacles are introduced. The main goal is to optimize the mixer by investigating the effects of obstruction geometry and shape. Numerical simulation is performed to compute fluid flow, fluid concentration, mixing index, and pressure drop by ANSYS Fluent 15 for  $1 \leq Re \leq 100$ . To validate the simulation setup, numerical data is compared with published experimental results. Finally, the best-performing mixer is proposed based on the overall performance.

## 2. Design of Micromixers

The geometry of an O mixer is investigated by M. Nimafar et al. [31] as shown in Figure 1. The inlet

channels and output channels present a square cross-section with an aspect ratio of 1:1 ( $\frac{W}{H} = \frac{0.4}{0.4} = 1$ ).

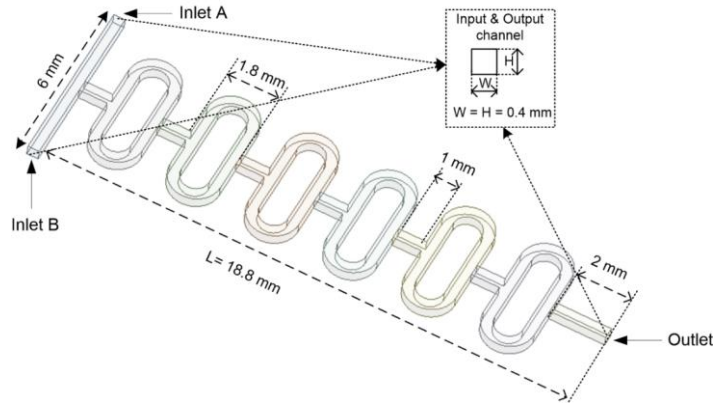


Figure 1. Diagram of a simple O mixer.

To enhance the mixing performance, circular-shaped obstacles and a combination of circular & cylindrical fin-shaped obstacles are placed inside the O mixer. The diameter of the circular-shaped obstacle ( $d$ ) is 0.2 mm. The length and width of the cylindrical fin obstacle ( $s$ ) are 0.3 mm and 0.07 mm, respectively. All examined mixers consist of 6 identical elements connected one after another and the total length is 18.8 mm (one element is 2.8 mm long). The detailed configuration of the obstacles in the O mixer is represented in Figure 2.

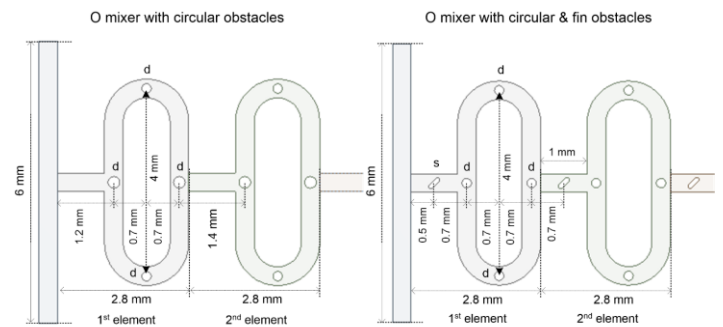


Figure 2. Diagram of the O mixer with circular obstacles, and the O mixer with circular & fin obstacles.

## 3. Numerical Setup and Methodology

The fluid behavior was studied using the ANSYS Fluent 15 commercial CED software. The fluid is considered incompressible, with steady-state, isothermal, and laminar flow conditions. The following advection-diffusion, Navier-Stokes, and continuity equations are used to solve the flow field: [42-43]

$$\nabla \cdot V = 0 \quad (1)$$

$$\rho V \nabla \cdot V = -\nabla P + \mu \nabla^2 V \quad (2)$$

$$V \cdot \nabla C = D \nabla^2 C \quad (3)$$

Where  $V$  is the fluid velocity ( $m/sec$ ),  $\rho$  is the fluid density ( $Kg/m^3$ ),  $P$  is the fluid pressure ( $Pa$ ),  $\mu$  is the fluid viscosity ( $\frac{Kg}{m \cdot sec}$ ),  $C$  is the fluid molar concentration ( $mol/m^3$ ), and  $D$  is fluid diffusivity ( $m^2/sec$ ).

The output of the mixer channel was set to zero (0) gauge pressure, the uniform flow velocity was used at both inlets (Inlet A and Inlet B), and the no-slip velocity condition was taken into account at walls for the numerical simulation. Relative species concentrations of fluids are assumed to be one (1) for inlet A and zero (0) for inlet B. The two input fluids are assumed to have the same density  $\rho = 998.2 \text{ Kg/m}^3$ , same dynamics viscosity  $\mu = 0.001 \text{ Pa s}$  and diffusivity  $D = 1 \times 10^{-9} \text{ m}^2/s$  [5]. The method known as SIMPLE (Semi Implicit Method for Pressure Equations) is utilized to solve the pressure-velocity coupling [7]. On the other hand, momentum and species concentration are handled using a second-order upwind strategy. In addition, scaled residuals of  $1 \times 10^{-6}$  are utilized to apply the convergence requirements for continuity, momentum, and species transport equations. Reynolds number is a crucial dimensionless parameter that is calculated by the following equation [44].

$$Re = \frac{\rho V d}{\mu} \quad (4)$$

$$d = \frac{2WH}{W + H} \quad (5)$$

Where  $Re$  is the Reynolds number,  $d$  is the characteristics hydraulic diameter ( $m$ ),  $W$  is the width of the mixing channel ( $m$ ) and  $H$  is the height of the mixing channel ( $m$ ). The mixing index is calculated using the following equations [45]

$$\sigma = \sqrt{\frac{1}{N} \sum_{i=1}^N (C_i - C_{av})^2} \quad (6)$$

$$\eta = 1 - \frac{\sigma}{\sigma_{max}} \quad (7)$$

Where,  $C_i$  is the concentration at the  $i^{\text{th}}$  node,  $C_{av}$  is the average concentration,  $\sigma$  is the standard deviation,

and  $\sigma_{max}$  is the maximum standard deviation ( $\sigma_{max} = 0.5$ ). Maximum and minimum efficiency can be zero ( $\eta = 0$ ) and one ( $\eta = 1$ ), respectively.

Mixing efficiency and pressure drop alone is not sufficient to have a complete comparison among various mixers. Hence, the mixing cost is computed by using pumping power which is used to flow liquid inside the mixer by following the equation [12-13].

$$M_C = \frac{\Delta P Q}{\eta} \quad (8)$$

Where  $M_C$  denotes the mixing cost ( $Watt$ ),  $\Delta P$  is the pressure drops ( $Pa$ ),  $Q$  is the flow rate ( $m^3/s$ ).

### 3. 1. Mesh Independence

Grid independence tests were conducted to determine an appropriate number of grids because the numerical results are always dependent on the mesh system. Fluent 15 was used to generate the structured grids with hexahedral elements for all mixers. An illustration of the O mixer with circular and fin obstacles is shown in Figure 3.

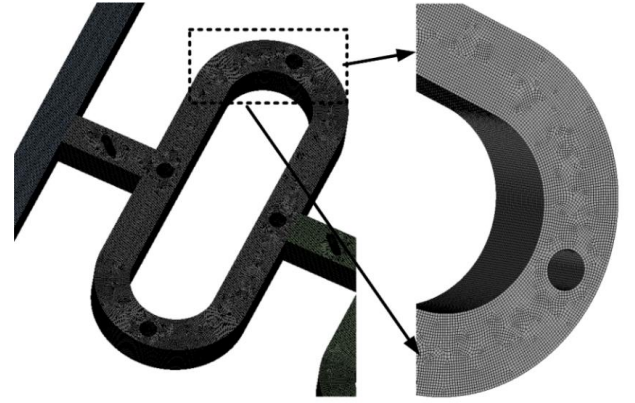


Figure 3. Hexagonal grids inside the O mixer with circular & fin obstacles.

The grid dependency of the O mixer with fin and circular obstacles is displayed in Figure 4 for five distinct nodes ranging from  $2.20 \times 10^6$  to  $6.97 \times 10^6$ . Along the mixer's axial length, the standard deviation of fluid mixing was computed at  $Re = 50$ . As the number of grids rises the standard deviation increases as indicated by Figure 5. The difference in standard deviation between nodes  $5.48 \times 10^6$  and  $6.97 \times 10^6$  is small. Thus, to obtain acceptable numerical data at a reasonable cost,  $5.48 \times 10^6$  nodes are utilized for simulation. Similarly, for the O mixer and the O mixer with circular obstacles,

$5.41 \times 10^6$  nodes and  $5.43 \times 10^6$  nodes of mesh were employed, respectively.

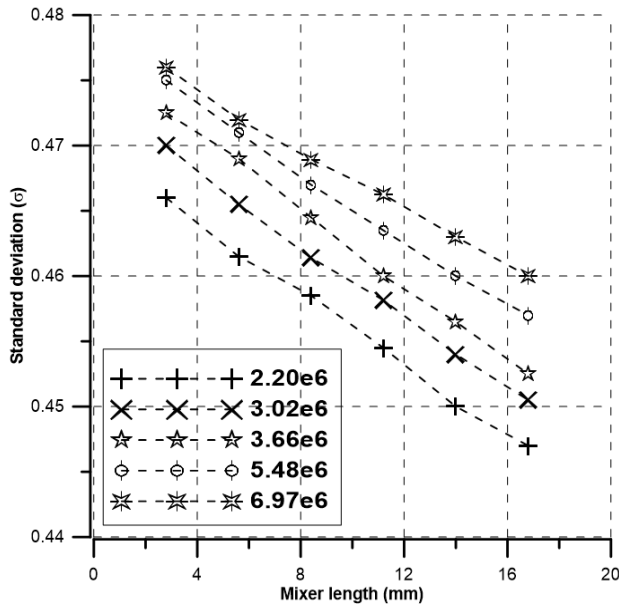
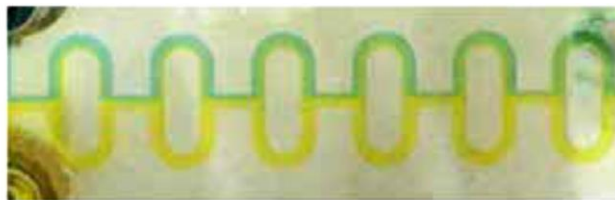


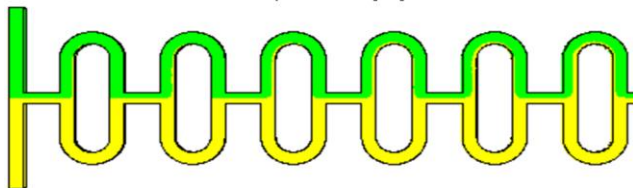
Figure 4. The standard deviation of the O mixer with circular & fin obstacles along the axial length at  $Re = 50$ .

### 3. 2. Numerical Validation

The numerical data obtained in this work is compared with published experimental results by M. Nimafar et al. [31] to verify the numerical setup. The top view of the fluid mixing concentration for the experimental test and computational model used in this study is demonstrated in Figure 5 at a Reynolds number of 4.166. Blue and yellow colors indicate two input fluids and two fluids mixed along the channel. The numerical finding, which is consistent with the experimental value, indicates poor species mixing.



Experimental [31]



Numerical

Figure 5. Top view of fluids concentration of the O mixer in experimental setup [31] and simulation at  $Re = 4.166$

As shown in Figure 6, a qualitative comparison is made between the numerical mixing index and the published experimental result [31] of the O micromixer without any obstacles. The maximum difference between experimental and numerical values is less than 20%. There are two potential causes for the disparity. First off, the computational model and the experimental prototypes utilized by M. Nimafar et al. [31] may differ in dimension and smoothness. Secondly, the numerical mixing efficiency is larger than the experimental values, which can be explained by the fact that the experimental efficiency was calculated using top-view images, whereas the numerical data was computed at the cross-sections of the channel.

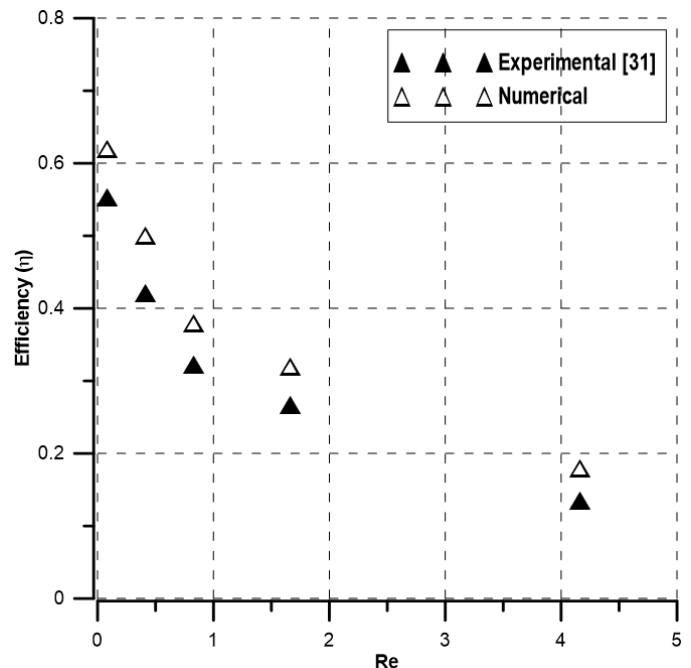


Figure 6. Comparison of numerical data and experimental value [31] of the O mixer after 15 mm along the axial length.

### 4. Result and Discussion

The mixing index of the three mixers was computed numerically for Reynolds numbers ranging from 1 to 100. The numerical mixing efficiency obtained for all three mixers is presented in Figure 7 and water distribution along the channel is illustrated in Figure 8. All mixers show an efficiency of more than 50% at  $Re \leq 1$ . At low Reynolds numbers ( $Re \leq 1$ ), molecular diffusion dominant the mixing process, and fluids have longer residence times resulting in high mixing efficiency. For channels with micromixers, at moderate Reynolds numbers ( $1 < Re \leq 10$ ), fluids have less time to mix resulting in poor efficiency as evident in Figure 7



and Figure 8. Efficiency starts to increase as the Reynolds numbers increase ( $Re > 10$ ). In this case, the mixing time decreases with the increase of Reynolds numbers (flow rate), but the fluid path becomes longer due to the split and recombination of fluids which compensates for the shorter mixing time as shown in Figure 8. This effect is more evident in the case of the O mixer with circular & fin obstacles and mixing efficiency is the highest (about 50%). Whereas the efficiency is about 15% and 20% for the O mixer and the O mixer with circular obstacles at  $Re = 100$ , respectively. In addition, the O mixer with circular & fin obstacles yields three times more efficiency than the other two at all examined Reynolds numbers.

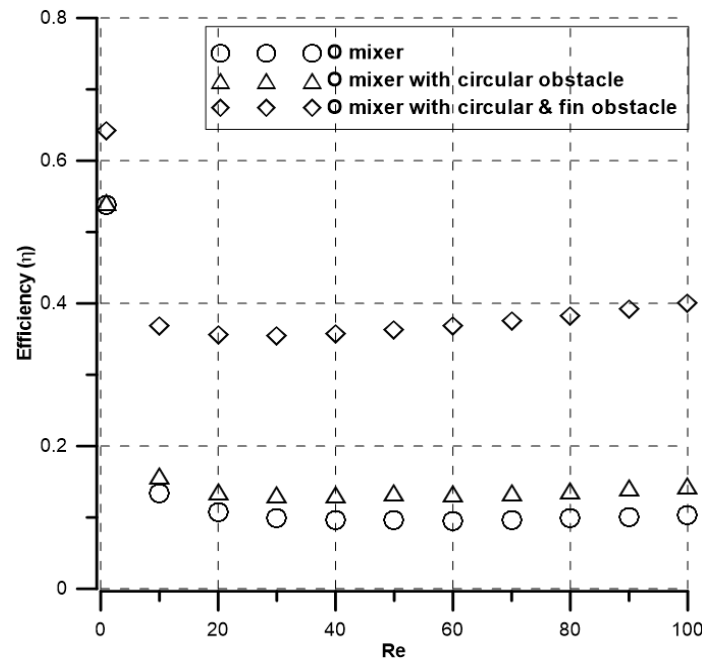


Figure 7. Numerical mixing efficiency at the output of the O mixer, the O mixer with circular obstacle, and the O mixer with circular & fin obstacle at varying Reynolds numbers.

Figure 9 presents the relationship between pressure drop, flow rate, and Reynolds numbers for all three mixers. Pressure drop increases with the increase of corresponding Reynolds numbers and flow rates. The O mixer shows the lowest pressure drop due to a lack of barriers and the O mixer with circular & fin obstacles produces the highest pressure drop. The O mixer with circular obstacles has a 1.5 times higher pressure drop than the O mixer but a 1.5 times lower pressure drop than the O mixer with circular & fin obstacles at  $Re = 100$ .

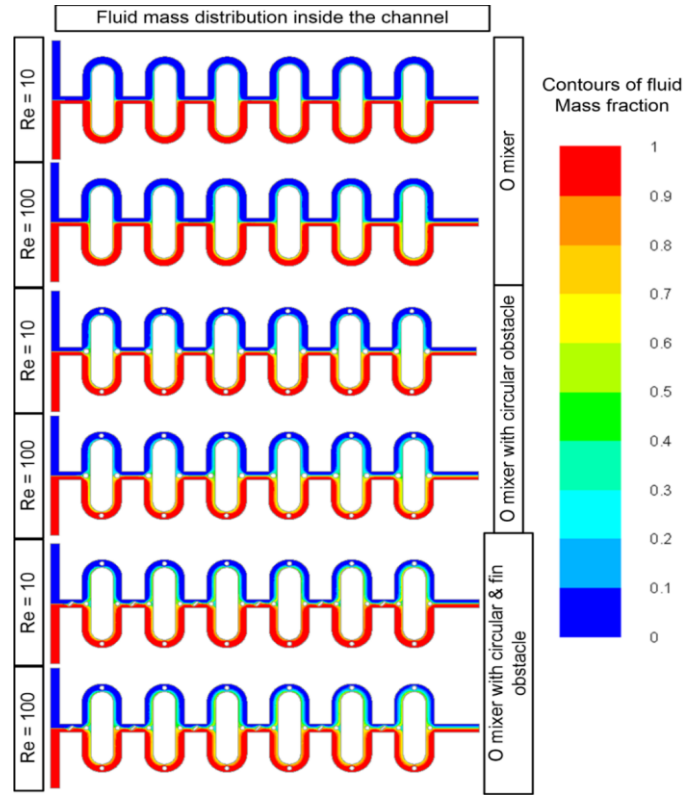


Figure 8. Fluid mass fraction contours at a mid-plane of the O mixer, the O mixer with circular obstacle, and the O mixer with circular & fin obstacle.

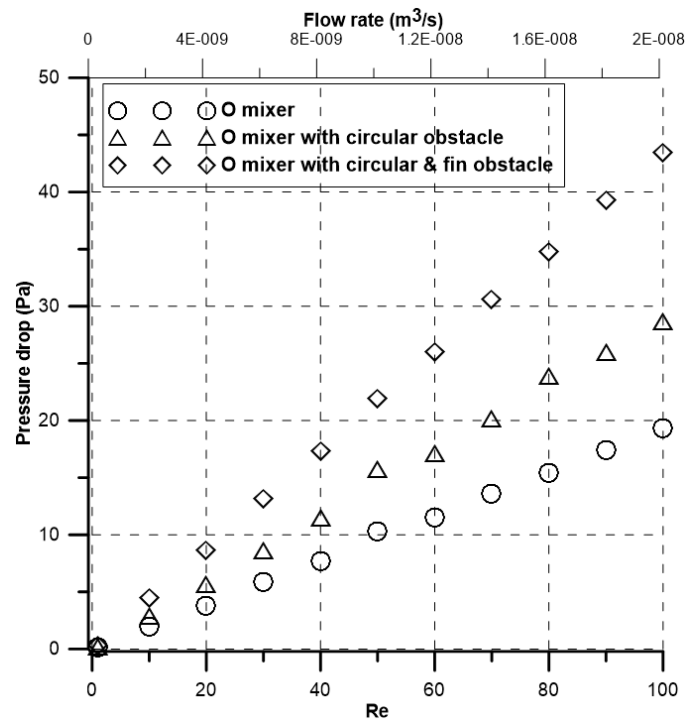


Figure 9. Numerical Pressure drop of the O mixer, the O mixer with circular obstacle, and the O mixer with circular & fin obstacle at varying Reynolds numbers and flow rate.

Mixing efficiency and pressure drop alone is not sufficient to suggest the best-performing mixer. Therefore, a parameter called mixing cost is evaluated and presented in Figure 10. The O mixer with circular obstacles shows the highest mixing cost. Whereas the mixing cost of the O mixer with circular & fin obstacles is the lowest despite the presence of the obstacles. Therefore, the O mixer with circular & fin obstacles is the best-performing mixer with the highest mixing efficacy and the lowest mixing cost.

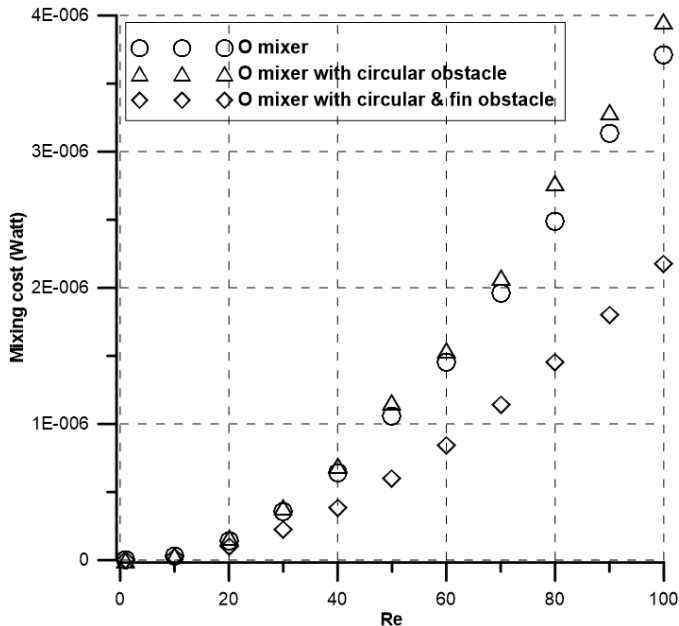


Figure 10. Numerical mixing cost of the O mixer, the O mixer with circular obstacle, and the O mixer with circular & fin obstacle at varying Reynolds numbers.

## 5. Conclusion

In this study, two types of obstacles (circular obstacles and circular & fin obstacles) are introduced to a planar O mixer. Numerical simulation was performed to evaluate the effect of obstacles on fluid flow, mixing performance, and pressure drop using the ANSYS 15 commercial software at a wide range of Reynolds numbers ( $1 \leq Re \leq 100$ ). The numerical results showed good agreement with the experimental and present good mixing performance over a wide range of flow conditions, particularly in the low  $Re$  ( $Re = 1$ ) and high  $Re$  ( $Re = 100$ ). The presented design is planar, and obstructions are all full channel height, thus can be constructed in a simple fabrication process. The introduction of circular and fin obstacles inside the O mixer increases the efficacy three times compared to the O mixer without obstacles, and a maximum of 50%

efficiency can be achieved with only six elements. Efficiency also increases with the increase of elements for all three mixers. Desire efficiency can be obtained by adding elements in series. Though the pressure drop of the O mixer with circular and fin obstacles is high, the mixing cost is the lowest due to its high efficacy. Finally, it can be proposed that the O micromixer with circular and fin obstacles is the best performing one which can be easily realized and integrated with microfluidic systems due to the simple planar structure.

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